

## Lagrangian analysis of the “swimmer” dynamical system

### Model of friction

Given a small one-dimensional stick of length  $d\lambda$  in a two-dimensional fluid medium, we postulate two forces that act upon it: a “viscous” friction force normal to the direction of the stick given by  $d\vec{F}_n = -k_1 \vec{n}(\vec{n} \cdot \vec{v})d\lambda$ , and a “laminar” friction force tangent to the direction of the stick given by  $d\vec{F}_t = -k_2 \vec{t}(\vec{t} \cdot \vec{v})d\lambda$ , where  $\vec{t}$ ,  $\vec{n}$  and  $\vec{v}$  are the tangent, normal, and velocity vectors.

Given a stick of length  $l$ , we can decompose its velocity into the linear center-of-mass velocity and the angular velocity relative to the cm:  $\vec{v}(\lambda) = \vec{v}_{cm} + \lambda \vec{n} \dot{\theta}$ , where  $\lambda \in [-\frac{l}{2}, \frac{l}{2}]$ . The normal and tangential frictional forces resulting from the linear part will be  $\vec{F}_n = -k_1 \vec{n}(\vec{n} \cdot \vec{v}_{cm})l$  and  $\vec{F}_t = -k_2 \vec{t}(\vec{t} \cdot \vec{v}_{cm})l$ , while the total moment arising from the angular part will be:

$$\tau = \int_{-\frac{l}{2}}^{\frac{l}{2}} \lambda \cdot df = \int_{-\frac{l}{2}}^{\frac{l}{2}} \lambda \cdot (-k_1 \lambda \dot{\theta}) d\lambda = -\frac{k_1}{12} l^3 \dot{\theta}$$

Later we will derive these forces from the dissipation function, which is a measure of the rate of energy being lost to friction.

### Generalized coordinates

We now connect  $k$  sticks together, and choose our generalized coordinates  $\mathbf{q}$  to be the location of the center of mass and the angles of the sticks relative to the  $x$  direction.

$$\boldsymbol{\theta} = [\theta_1 \quad \theta_2 \quad \dots \quad \theta_k]^T \quad \mathbf{q} = \begin{bmatrix} \vec{r}_{cmx} \\ \vec{r}_{cm y} \\ \boldsymbol{\theta} \end{bmatrix}$$

We define the unit tangent and normal vectors,

$$\vec{t}_i = \begin{bmatrix} \cos(\theta_i) \\ \sin(\theta_i) \end{bmatrix} \quad \vec{n}_i = \begin{bmatrix} -\sin(\theta_i) \\ \cos(\theta_i) \end{bmatrix}$$

Defining  $\tilde{r}_i$  to be the location of the  $i$ th stick relative to the center-of-mass, we can now write the equations that relate the  $\tilde{r}_i$  's to the  $\theta_i$  's:

$$\begin{bmatrix} \tilde{r}_{i+1} - \tilde{r}_i = \frac{l_i}{2} \vec{t}_i + \frac{l_{i+1}}{2} \vec{t}_{i+1} \\ \sum m_i \tilde{r}_i = 0 \end{bmatrix}$$

The first  $k-1$  equation can be understood from Figure 1, while the last equation stems from the  $\tilde{r}_i$  's being defined in the frame of the center-of-mass.

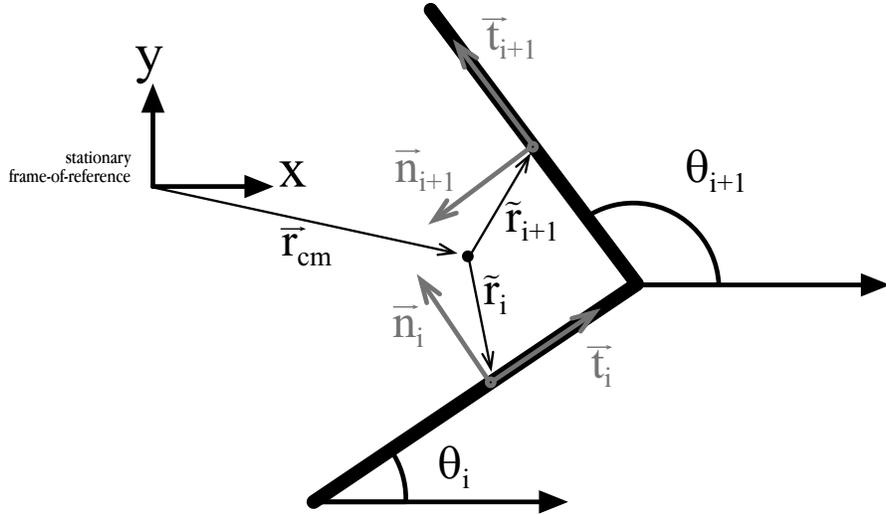


Figure 1

By taking the derivative with respect to time, we get the equations which give the velocities relative to the center-of-mass:

$$(1) \quad \left[ \begin{array}{l} \tilde{v}_{i+1} - \tilde{v}_i = \frac{l_i}{2} \tilde{n}_i \dot{\theta}_i + \frac{l_{i+1}}{2} \tilde{n}_{i+1} \dot{\theta}_{i+1} \\ \sum_i m_i \tilde{v}_i = 0 \end{array} \right]$$

### Dynamics

We are now in a position to write the equations of the dynamics. The Lagrangian  $L$  has only the kinetic energy component and is given by

$$L = \frac{1}{2} \vec{v}_{cm}^2 \sum_i m_i + \frac{1}{2} \sum_i m_i \tilde{v}_i^2 + \frac{1}{2} \sum_i I_i \dot{\theta}_i^2$$

Where  $I_i$  is the  $i$ th moment-of-inertia  $I_i = \frac{1}{12} m_i l_i^2$ . The dissipation function is given by

$$F = \frac{1}{2} k_1 \sum_i [l_i (\tilde{v}_i \cdot \tilde{n}_i)^2 + \frac{l_i^3}{12} \dot{\theta}_i^2] + \frac{1}{2} k_2 \sum_i l_i (\tilde{v}_i \cdot \tilde{t}_i)^2$$

Note that the velocities in the dissipation function are relative to the fluid, i.e.  $\vec{v}_i = \tilde{v}_i + \vec{v}_{cm}$ . The dynamics are now given by the E-L equations

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_i} \right) - \frac{\partial L}{\partial q_i} + \frac{\partial F}{\partial \dot{q}_i} = 0$$

The E-L equations for the center-of-mass are given simply by

$$\dot{\vec{v}}_{cm} \sum_i m_i + k_1 \sum_i (\tilde{v}_i \cdot \tilde{n}_i) \tilde{n}_i + k_2 \sum_i (\tilde{v}_i \cdot \tilde{t}_i) \tilde{t}_i = 0$$

However, for the angular variables  $\theta_i$ , we will require some better notation.

### Matrix definitions

We now define several vectors and matrices which will help us reformulate the problem in a more convenient and legible notation. First we define a set of diagonal matrices:

$$\begin{aligned}\Theta &= \text{diag}(\boldsymbol{\theta}) & \mathbf{L} &= \text{diag}(l_i) \\ \mathbf{M} &= \text{diag}(m_i) & \mathbf{I} &= \frac{1}{12} \mathbf{M} \mathbf{L}^2 \\ \mathbf{T}_x &= \text{diag}(\cos(\theta_i)) & \mathbf{T}_y &= \text{diag}(\sin(\theta_i)) \\ \mathbf{N}_x &= \text{diag}(-\sin(\theta_i)) & \mathbf{N}_y &= \text{diag}(\cos(\theta_i))\end{aligned}$$

Note that for both the  $x$  and  $y$  dimensions  $\dot{\mathbf{N}} = -\mathbf{T}\dot{\Theta}$ . Now, defining the relative and absolute velocity vectors,

$$\tilde{\mathbf{v}}_x = \begin{bmatrix} \tilde{v}_{1,x} \\ \tilde{v}_{2,x} \\ \vdots \\ \tilde{v}_{k,x} \end{bmatrix} \quad \tilde{\mathbf{v}}_y = \begin{bmatrix} \tilde{v}_{1,y} \\ \tilde{v}_{2,y} \\ \vdots \\ \tilde{v}_{k,y} \end{bmatrix} \quad \mathbf{v}_x = \begin{bmatrix} \tilde{v}_{1,x} + v_{cmx} \\ \tilde{v}_{2,x} + v_{cmx} \\ \vdots \\ \tilde{v}_{k,x} + v_{cmx} \end{bmatrix} \quad \mathbf{v}_y = \begin{bmatrix} \tilde{v}_{1,y} + v_{cm y} \\ \tilde{v}_{2,y} + v_{cm y} \\ \vdots \\ \tilde{v}_{k,y} + v_{cm y} \end{bmatrix}$$

and the coefficient matrices,

$$\mathbf{Q} = \begin{bmatrix} -1 & 1 & & & \\ & -1 & 1 & & \\ & & -1 & 1 & \\ & & & \ddots & \ddots \\ m_1 & m_2 & m_3 & \cdots & m_k \end{bmatrix} \quad \mathbf{A} = \begin{bmatrix} 1 & 1 & & & \\ & 1 & 1 & & \\ & & \ddots & \ddots & \\ 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

we can now rewrite equations (1) as

$$\begin{aligned}\mathbf{Q}\tilde{\mathbf{v}}_x &= \frac{1}{2} \mathbf{A} \mathbf{L} \mathbf{N}_x \dot{\boldsymbol{\theta}} & \tilde{\mathbf{v}}_x &= \mathbf{P} \mathbf{N}_x \dot{\boldsymbol{\theta}} \\ \mathbf{Q}\tilde{\mathbf{v}}_y &= \frac{1}{2} \mathbf{A} \mathbf{L} \mathbf{N}_y \dot{\boldsymbol{\theta}} & \tilde{\mathbf{v}}_y &= \mathbf{P} \mathbf{N}_y \dot{\boldsymbol{\theta}}\end{aligned} \quad \text{with} \quad \mathbf{P} = \frac{1}{2} \mathbf{Q}^{-1} \mathbf{A} \mathbf{L}$$

So that  $\frac{\partial}{\partial \boldsymbol{\theta}} \tilde{\mathbf{v}}_n = (\mathbf{P} \mathbf{N}_x)^T = \mathbf{N}_x \mathbf{P}^T$ , and similarly for the tangential velocities.

We will additionally define the normal and tangential velocity vectors (relative to the fluid) as

$$\mathbf{v}_n = \begin{bmatrix} (\tilde{v}_1 + \tilde{v}_{cm}) \tilde{n}_1 \\ (\tilde{v}_2 + \tilde{v}_{cm}) \tilde{n}_2 \\ \vdots \\ (\tilde{v}_k + \tilde{v}_{cm}) \tilde{n}_k \end{bmatrix} = \mathbf{N}_x \mathbf{v}_x + \mathbf{N}_y \mathbf{v}_y \quad \mathbf{v}_t = \begin{bmatrix} (\tilde{v}_1 + \tilde{v}_{cm}) \tilde{t}_1 \\ (\tilde{v}_2 + \tilde{v}_{cm}) \tilde{t}_2 \\ \vdots \\ (\tilde{v}_k + \tilde{v}_{cm}) \tilde{t}_k \end{bmatrix} = \mathbf{T}_x \mathbf{v}_x + \mathbf{T}_y \mathbf{v}_y$$

Also, it will be found convenient to define the constant matrix

$$\mathbf{G} = \mathbf{P}^T \mathbf{M} \mathbf{P}$$

### The $\theta$ dynamics

The Lagrangian is now given by

$$\begin{aligned}L &= \frac{1}{2} \tilde{v}_{cm}^2 \text{tr}(\mathbf{M}) + \frac{1}{2} \tilde{\mathbf{v}}_x^T \mathbf{M} \tilde{\mathbf{v}}_x + \frac{1}{2} \tilde{\mathbf{v}}_y^T \mathbf{M} \tilde{\mathbf{v}}_y + \frac{1}{2} \dot{\boldsymbol{\theta}}^T \mathbf{I} \dot{\boldsymbol{\theta}} \\ &= \frac{1}{2} \tilde{v}_{cm}^2 \text{tr}(\mathbf{M}) + \frac{1}{2} \dot{\boldsymbol{\theta}}^T (\mathbf{N}_x \mathbf{G} \mathbf{N}_x + \mathbf{N}_y \mathbf{G} \mathbf{N}_y + \mathbf{I}) \dot{\boldsymbol{\theta}}\end{aligned}$$

The generalized momenta for the  $\theta$  coordinates are

$$\frac{\partial}{\partial \dot{\boldsymbol{\theta}}} L = [\mathbf{I} + \mathbf{N}_x \mathbf{G} \mathbf{N}_x + \mathbf{N}_y \mathbf{G} \mathbf{N}_y] \dot{\boldsymbol{\theta}}$$

and their time derivatives

$$\frac{d}{dt} \left( \frac{\partial}{\partial \dot{\boldsymbol{\theta}}} L \right) = [\mathbf{I} + \mathbf{N}_x \mathbf{G} \mathbf{N}_x + \mathbf{N}_y \mathbf{G} \mathbf{N}_y] \ddot{\boldsymbol{\theta}} - 2[\mathbf{N}_x \mathbf{G} \mathbf{T}_x + \mathbf{N}_y \mathbf{G} \mathbf{T}_y] \dot{\boldsymbol{\theta}}^2$$

Where  $\dot{\boldsymbol{\theta}}^2 = \dot{\boldsymbol{\theta}}^T \dot{\boldsymbol{\theta}}$  is the vector of squared velocities.

The partial derivative with respect to the  $\theta$ s is identically zero:  $\frac{\partial}{\partial \boldsymbol{\theta}} L = 0$ .

The dissipation function is given by

$$F = \frac{1}{2} k_1 (\mathbf{v}_n^T \mathbf{L} \mathbf{v}_n + \frac{1}{12} \dot{\boldsymbol{\theta}}^T \mathbf{L}^3 \dot{\boldsymbol{\theta}}) + \frac{1}{2} k_2 \mathbf{v}_t^T \mathbf{L} \mathbf{v}_t$$

And since

$$\frac{\partial}{\partial \boldsymbol{\theta}} \mathbf{v}_n = \frac{\partial}{\partial \boldsymbol{\theta}} (\mathbf{N}_x \tilde{\mathbf{v}}_x + \mathbf{N}_y \tilde{\mathbf{v}}_y) = \mathbf{N}_x \mathbf{P}^T \mathbf{N}_x + \mathbf{N}_y \mathbf{P}^T \mathbf{N}_y$$

$$\frac{\partial}{\partial \boldsymbol{\theta}} \mathbf{v}_t = \frac{\partial}{\partial \boldsymbol{\theta}} (\mathbf{T}_x \tilde{\mathbf{v}}_x + \mathbf{T}_y \tilde{\mathbf{v}}_y) = \mathbf{N}_x \mathbf{P}^T \mathbf{T}_x + \mathbf{N}_y \mathbf{P}^T \mathbf{T}_y$$

the generalized forces are

$$\frac{\partial}{\partial \boldsymbol{\theta}} F = k_1 (\mathbf{N}_x \mathbf{P}^T \mathbf{N}_x + \mathbf{N}_y \mathbf{P}^T \mathbf{N}_y) \mathbf{L} \mathbf{v}_n + \frac{k_1}{12} \mathbf{L}^3 \dot{\boldsymbol{\theta}} + k_2 (\mathbf{N}_x \mathbf{P}^T \mathbf{T}_x + \mathbf{N}_y \mathbf{P}^T \mathbf{T}_y) \mathbf{L} \mathbf{v}_t$$

So that finally the dynamics are given by solving

$$\frac{d}{dt} \left( \frac{\partial}{\partial \dot{\boldsymbol{\theta}}} L \right) - \frac{\partial}{\partial \boldsymbol{\theta}} L + \frac{\partial}{\partial \boldsymbol{\theta}} F = \begin{bmatrix} u_1 \\ u_2 - u_1 \\ \vdots \\ u_{k-1} - u_{k-2} \\ -u_{k-1} \end{bmatrix}$$

for  $\ddot{\boldsymbol{\theta}}$ .